Amendments to the Claims:

This listing of claims will replace all prior versions, and listings, of claims in the application:

Listing of Claims:

1. **(Currently Amended)** A method for operating an air conditioning system ~~(1)~~ of a vehicle, **comprising:** ~~**in which a fluid (F) for conditioning an air stream (2) is circulated in a circuit (8) operable in the cooling or heating mode, characterized in that, in the heating mode, the circuit comprises a condenser (26), a heat exchanger (24) and an intermediate store (28), the circuit being controlled in such a way**~~

circulating in a circuit a fluid to condition an airstream;

operating the circuit in a cooling mode or a heating mode, wherein the circuit includes a condenser or a compressor, a heat exchanger, and an intermediate store in the heating mode; and

controlling the circuit such that the intake pressure of the condenser **or the compressor** ~~(26)~~ at least partially overshoots a saturation pressure in the circuit caused by the ambient temperature.

2. **(Currently Amended)** The method as claimed in claim 1, ~~**characterized in that the heating mode corresponds to an operation of the circuit**~~ **further comprising operating the circuit during the heating mode** in a dextrorotary triangulation process, ~~**in which the**~~ **wherein an** exit **side** of the condenser **or the compressor connects to an entry side** ~~**is connected to an entry**~~ of a control valve ~~(38a)~~, **an exit side of the control valve connects to an entry side of the heat exchanger, an exit side of the heat exchanger connects to an entry side of the intermediate store, and an exit side of the intermediate store connects to an entry side** ~~**connected on the exit side to the heat exchanger (24) which is followed on the exit side by the intermediate store (28) and the entry**~~ of the condenser **or the compressor** ~~(26)~~.

3. **(Currently Amended)** The method as claimed in claim 1, ~~**characterized in that the**~~ **further comprising controlling** intake pressure ~~**can be controlled**~~ in a range of 10 bar to 110 bar.

4. (**Currently Amended**) The method as claimed in claim 1, ~~**characterized in that, in the heating mode,**~~ **further comprising dividing** the fluid ~~**(F)**~~ in the circuit ~~**can be divided**~~ into at least one active part ~~**(8B)**~~ and at least one passive part ~~**(8A)**~~ **while in the heating mode**.

5. (**Currently Amended**) The method as claimed in claim 1, ~~**characterized in that, with the activation of**~~ **further comprising:**

activating the heating mode~~**; and, the fluid (F) is routed**~~

routing the fluid out of the passive part of the circuit ~~**(8A)**~~ into the active part of the circuit ~~**(8B)**~~.

6. (**Currently Amended**) The method as claimed in claim 1, ~~**characterized in that,**~~ **further comprising routing out the fluid of the passive part of the circuit into the active part of the circuit** when a predeterminable threshold value for the intake pressure in the active part of the circuit ~~**(8B)**~~ is undershot~~**, the fluid (F) is routed out of the passive part of the circuit (8A) into the active part of the circuit (8B)**~~.

7. (**Currently Amended**) The method as claimed in either claim 5, ~~**characterized in that,**~~ **further comprising transferring** ~~**to transfer**~~ the fluid out of the passive part of the circuit into the active part of the circuit **by changing the circuit operated in the heating mode over to the cooling mode**~~**, the circuit operated in the heating mode is changed over to the cooling mode**~~.

8. (**Currently Amended**) The method as claimed in either claim 5, ~~**characterized in that,**~~ **further comprising transferring** ~~**to transfer**~~ the fluid out of the passive part of the circuit into the active part of the circuit **by changing the circuit operated in the heating mode to a laevorotatory triangular process**~~**, the circuit operated in the heating mode is changed over to a laevorotatory triangulation process**~~.

9. **(Currently Amended)** The method as claimed in either claim 7, ~~**characterized in that**~~ **further comprising:**

operating the circuit ~~**can be operated**~~ in the cooling mode or in the laevorotatory triangulation process up to the undershooting of a settable threshold value**[[,]]; and**

changing the circuit ~~**being capable of being changed over**~~ to the heating mode again after the undershooting of the threshold value.

10. **(Currently Amended)** The method as claimed in claim 9, ~~**characterized in that**~~ **further comprising predetermining** the threshold value for an intake pressure and/or for a high pressure and/or for a hot-gas temperature at the condenser **or the compressor** ~~**can be predetermined**~~.

11. **(Currently Amended)** The method as claimed in claim 9, ~~**characterized in that**~~ **further comprising setting** the threshold value of the intake pressure ~~**is set**~~ at **at** least 3 bar~~, preferably 5 bar,~~ below the value of the saturation pressure caused by the ambient temperature.

12. **(Currently Amended)** The method as claimed in claim 7, ~~**characterized in that the circuit can be operated**~~ **further comprising:**

operating the circuit in the cooling mode or in the laevorotatory triangulation process for a predeterminable period of time**[[,]]; and**

changing the circuit ~~**being capable of being changed over**~~ to the heating mode ~~**again**~~ after ~~**the expiry**~~ **expiration** of the period of time.

13. **(Currently Amended)** The method as claimed in claim 7, ~~**characterized in that**~~ **further comprising reducing** an air stream ~~**(2)**~~ through the evaporator ~~**can be reduced**~~ after the changeover to the cooling mode or to the laevorotatory triangulation process.

14. **(Currently Amended)** The method as claimed in claim 7, ~~**characterized in that**~~ **further comprising reducing** an air stream through a gas cooler ~~**can be reduced**~~ after the changeover to the cooling mode or to the laevorotatory triangulation process.

15. **(Currently Amended)** The method as claimed in claim 10, ~~**characterized in that a pressure equalization can be carried out**~~ **further comprising equalizing pressure** in the circuit after ~~**the return**~~ **returning** to the heating mode.

16. **(Currently Amended)** An air conditioning system ~~**(1)**~~ for a vehicle ~~**with**~~ **comprising:**

a circuit ~~**(8)**~~, operable in **[[the]] a** cooling or heating mode, ~~**for the circulation of**~~ **configured to circulate** a fluid ~~**(F) for conditioning, the fluid configured to condition**~~ an air stream ~~**(2), characterized in that,**~~ **wherein** in the heating mode, the circuit ~~**comprises**~~ **includes:**

a heat exchanger ~~**(24)**~~,

an intermediate store ~~**(28)**~~, and

a condenser **or a compressor** ~~**(26)**~~ for the intermediate storage or for the condensation of the fluid ~~**(F)**~~,

wherein the condenser **or the compressor** ~~**being operated**~~ **is configured to operate** at an intake pressure ~~**which**~~ **that** is higher than the saturation pressure in the circuit ~~**(8)**~~ caused by the ambient temperature.

17. **(Currently Amended)** The air conditioning system ~~**(1)**~~ as claimed in claim 16, ~~**characterized in that the**~~ **further comprising:**

an evaporator, ~~**(6) inserted**~~

wherein a secondary side is included in **[[the]] a** flow duct ~~**(4)**~~ of the air stream ~~**(2)**~~ on ~~**the secondary side,**~~ **[[and]] a primary side is connected to [[in]]** the circuit ~~**(8) on the primary side is provided, which is connected in the circuit (8)**~~, **and an exit side is connected to the intermediate store** ~~**on the exit side, to the intermediate store (28)**~~, **and**

wherein ~~**with**~~ a nonreturn valve ~~**(36) being**~~ **is** interposed **between the evaporator and the intermediate store**.

18. **(Currently Amended)** The air conditioning system ~~(1)~~ as claimed in claim 17, ~~**characterized in that**~~ **wherein** the volume of the evaporator ~~(6)~~ for fluid reception is smaller than the storage volume of the intermediate store ~~(28)~~.

19. **(Currently Amended)** The air conditioning system ~~(1)~~ as claimed in claim 18, ~~**characterized in that**~~ **wherein** the ratio of the storage volume of the intermediate store to the volume of the evaporator lies in the range of 2:1 to 20:1~~, preferably in the range of between 2:1 and 10:1~~.

20. **(Currently Amended)** The air conditioning system ~~(1)~~ as claimed in claim 16, ~~**in which**~~ **further comprising** a control device ~~(38B) is~~ arranged between the heat exchanger ~~(24)~~ and the intermediate store ~~(28)~~.

21. **(Currently Amended)** The air conditioning system ~~(1)~~ as claimed in claim 16, ~~**in which**~~ **further comprising** a pressure sensor **[[is]]** assigned on the intake side to the condenser **or the compressor** ~~(26)~~.

22. **(Currently Amended)** The air conditioning system ~~(1)~~ as claimed in claim 16, ~~**in which**~~ **wherein** the circuit ~~(8)~~ is subdivided into at least one active **part** and at least one passive part.

23. **(Currently Amended)** The air conditioning system ~~(1)~~ as claimed in claim 22, ~~**in which**~~ **wherein** the active part is connected to the passive part by ~~**means of**~~ a ~~**further**~~ control device ~~(38C), the control device (38C) being opened~~ **configured to open** when the fluid quantity in the active part of the circuit overshoots a predeterminable threshold value.

24. **(Currently Amended)** The air conditioning system ~~(1)~~ as claimed in claim 19, ~~**in which**~~ **wherein** the condenser **or the compressor** ~~(26)~~ is connected to the evaporator ~~(6)~~ on the exit side via a control means ~~(42)~~ and on the entry side via an associated controllable

connecting line ~~(40)~~, after the opening of the control means gaseous fluid ~~(F) passing~~ **passes** into the evaporator and ~~forcing~~ **forces** liquid fluid ~~(F)~~ out of the evaporator into the active part ~~(8B)~~ of the circuit.

25. (**New**) The method as claimed in claim 11, further comprising setting the threshold value of the intake pressure at 5 bar below the value of the saturation pressure caused by the ambient temperature.

26. (**New**) The air conditioning system as claimed in claim 19, wherein the ratio of the storage volume of the intermediate store to the volume of the evaporator lies in the range of between 2:1 and 10:1.